

Theoretical Analysis on the Effect of Operation Strategies on the Environmental Performance of Micro Gas Turbine Trigeneration System in Tropical Region

KH Koo, MRR Chand, Hassan Ibrahim, Azizuddin Abd Aziz, Firdaus Basrawi*

Energy Sustainability Focus Group, Faculty of Mechanical, Universiti Malaysia Pahang, Pekan, Malaysia

Abstract Trigeneration system (TGS) is a type of distributed power generation system that can further utilize waste heat for heating and cooling. This will result in 60-90% of energy utilization efficiency. Micro Gas Turbine is one of attractive prime movers that can reduce environmental pollution, less maintenance, and flexible in fuel supply. However, environmental performance of MGT-TGS depends on the operation strategy employed. In order to reduce emission of CO₂, NO_x and CO, studied on optimum operation strategy based on environmental performance of MGT-TGS in residential building loads for tropical region was carried out. Two actual size of MGT, 30kW and 65kW were used as the prime mover, and equipment and configuration of MGT-TGS were designed to cover energy load of a residential building in tropical region. Emission reduction index was used to compare emissions of MGT-TGS with different operation strategies. Result shows that MGT with smaller capacity at higher operation load will produce least emissions. Thus, base load is the most optimum operation strategy in terms of environmental performance. Thus, the best solution is base load that used smaller MGT. However, economic performance also must be further studied because smaller MGT has higher cost per kW, and it has slightly lower power generation efficiency.

Keywords Micro Gas Turbine, Trigeneration system, Operation strategy, Emission

1. Introduction

In traditional power plant the electricity is produced by burning fossil fuel to drive the turbine and generator. This type of power plant can produce large amount of heat in the condenser and become waste. Normally the waste heat generated in conventional power plant is about 67% of the total fossil fuel content [1]. If power plant is located close to end-user, this waste heat can be utilized for space and water heating and also cooling, this type of system that can produce electricity, heat and cooling simultaneously is called Trigeneration System (TGS).

Another reason when considering that is in Malaysia around 7.5% of total electricity generation is lost during the transmission & distribution networks [2]. Due to this reason trigeneration can be used, and this will result in emission reduction, increase total energy efficiency, and potentially reducing energy costs.

TGS that produces power, heat and cooling has been thought to be a significant energy proficiency measure. Especially, small TGS frameworks, for example,

micro-TGSs are generally upheld by numerous governments, particularly in Northern Europe [3]. Micro-TGSs are seen as an option for delivering power supply with the extra capacity of providing residential heating capabilities, thus increase the overall energy proficiency of the system. A few studies say that its penetration could be imperative by 2050 [4].

Micro-TGSs have extra value of the flexibility to the user which conventional generation is impossible to do it. Most of these technologies have flexible in its size, expandability, and operation. There are different type of operating strategies that are commonly used to meet the on-site energy loads. Selection of appropriate operation strategy is very important during system design stage, and appropriate operation strategy depends on the power, heating and cooling load of a target building. In addition, part-load operation is also important because it drastically affect the performance of TGS. Main operating strategies of TGS are:

a) Power match mode

Micro-TGS is specially installed to meet electric load of the user. Excess of electricity can be sold back to grid. Additional boiler may need to cover heat demand of the user.

b) Heat match mode

Micro-TGS is specially installed to meet heating load of the user. Excess heat can be stored in additional equipment which is heat storage tank. User may need to buy electricity

* Corresponding author:

mfirdausb@ump.edu.my (Firdaus Basrawi)

Published online at <http://journal.sapub.org/jmea>

Copyright © 2015 Scientific & Academic Publishing. All Rights Reserved

from national grid to cover power demand.

c) Mix match mode

Micro-TGS is specially installed to meet both power and heating loads that cover the user demand in whole day. Excess or insufficient of power and heat may need additional equipment.

d) Base load match mode

Micro-TGS is installed to only cover base load. Thus, MGT operates at constant and full load and this will result in insufficient electricity and heat. The insufficient electricity can be bought from grid, the heating load can be stored in heat storage tank.

There are a few type of prime mover for TGS including fuel cell, reciprocating engine. MGT has some advantages over other prime movers. These advantages are: compact systems and lightweight, vibration free and low noise level, lowest emission, heat recovery system is simple, require less maintenance cost and time. Furthermore, it can be operated as: stand-alone or grid connected system to produce large amount of electricity.

Performance of MGT-TGS depend on operation strategy employed and pattern of building load. This operation that cover different type of building loads will lead to different operation loading of MGT-TGS, subsequently affect the emissions released from the system. Whether the MGT-TGS that is employed will be even worse, or better compared to a conventional system depends on the operation strategy employed.

There are many recent studies reported on the MGT-TGS. Baswari F, et al, studied on optimum operating strategies of a stand-alone hybrid-photovoltaic MGT-TGS [5]. Bogdan AT, et al, compared economic factor of biomass internal combustion engine and natural gas MGT by using payback period in Romania [6]. Uzuneanu K, et al, evaluate the energy performance and efficiency of trigeneration system using fuel cells [7]. Xu DH, et al, analyze energy, environmental and economic performance for MGT-TGS data centers [8]. Simon PB, et al, studied performance of MGT-TGS for residential buildings assessed on the basis of the system's energetic, environmental and economic performance [9]. Yu HD, et al, evaluate energy efficiency and emission of MGT-TGS using biofuel [10]. N Sugiarta, et al, evaluation of energy efficiency, economic and environmental performance of a MGT-TGS for supermarket applications in South England [11]. Wang JJ, et al, perform analysis on 3 criteria: primary energy saving, carbon dioxide reduction and annual total cost saving for MGT-TGS with electrical and heat demand in different climate zones of China [12]. Ebrahimi M, et al, analyze energy and exergy of micro-steam trigeneration system in residential building [13]. KC Kavvadias, et al, analyze economic performance of trigeneration system with different operation strategies for hospital building [14]. MA Lozano, et al, studied on optimum operation strategies of a simple trigeneration system by marginal cost as economic performance [15]. PJ Mago, et al, studied performance of hybrid electric-thermal

operating strategies of MGT-TGS on primary and site energy consumption, carbon dioxide reduction and operational cost [16]. From the literature review, there is no finding reported on the effect of different operation strategies on the environmental performance of MGT-TGS for a residential building in tropical region.

Thus, the objective of this study is to determine optimum operation strategy based on environmental performance of MGT-TGS in residential building in tropical region. Since, power, heating and cooling load are commonly used in tropical region, 4 different operation strategies of MGT-TGS with different equipment were design to cover these building loads. These strategies are power match, heat match, mix match and base load. MGT with different capacity of 30 kW and 65 kW were used as main component that produce power. All the equipment in each operation strategies was clarified by energy balance calculation. Emission reduction index of CO₂, NO_x and CO emission were compared with each strategies. Finally, the best strategy is chosen based on the emission reduction index.

2. Materials and Methods

2.1. Micro Gas Turbine Trigeneration System Configuration

Schematic diagram below shows the system configuration of the MGT-TGS studied [5]. Core component of the system is MGT 30 kW and 65 kW. Both sizes of MGT are commonly available for residential used. In power production, MGT covers the power demand. In heat production, exhaust heat recovered is used to cover water heating demand and to operate an absorption heat pump. Absorption heat pump will convert heat energy to cooling load and cover cooling demand.

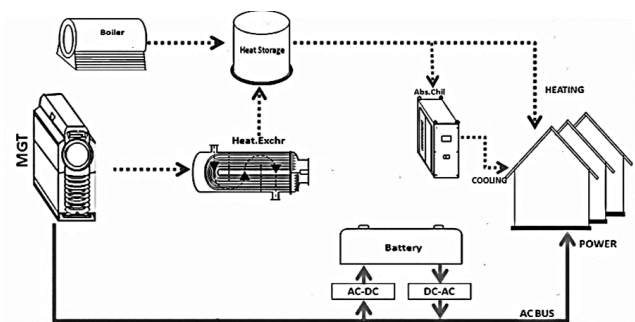


Figure 1. Configuration of MGT System for All Operation Strategy

In power match operation, MGT loading is adjusted to cover the power demand of the building, heat match is adjusted to cover heat demand, mix match cover both power and heat demand, base load is operated with constant full load by cover base power demand. Insufficient power was bought from TNB, meanwhile heat storage was used to balance the heating demand during day and night time. In addition, boiler was used to supply insufficient heat, and excess of power produced was stored in battery.

MGT is usually designed to have optimum energy and

emissions performance at full load. Under partial load operation, its power generation efficiency decreases and its emissions level also increases rapidly. Thus, base load strategies with smaller MGT capacity is used to run constantly with full load. Peak demand can be covered by top up from external grid and addition of boiler is used to cover insufficient of heat supply. This operation is expected to have lowest emission but might subject to high cost due to extra power top up from external grid.

2.2. Ambient Temperature Condition

Ambient temperature is important as the input of the calculation for MGT. Ambient temperature of Malaysia is used. As a tropical climate country, the different of the highest and lowest ambient temperature range of Malaysia is between 33°C and 24°C with average of 28°C [5].

2.3. Data of Energy Demand and Residential Building

A reference building is used to study the energy demands. This building is the most commonly built in Malaysia and the studied building is a group of 148 terrace houses. Each house is 6.5 m long and 19.8 m wide. It has four rooms and bathrooms that cover a total area of 129 m² [5]. Data for energy load for each house was adopted from a survey. Power, heating and cooling demand of a single house are shown in Figure 1 [5]. Power demand increase during day hours whereas cooling demand increase during night time because residential use air conditioner during night time and less power is used at that period. Besides building demands, a particular length of water pipeline and power capacity of water pumps were also needed to calculate heat losses throughout the pipeline and power for water pumping.

Amount of energy demands for 148 houses were assumed to be same using average energy demands.

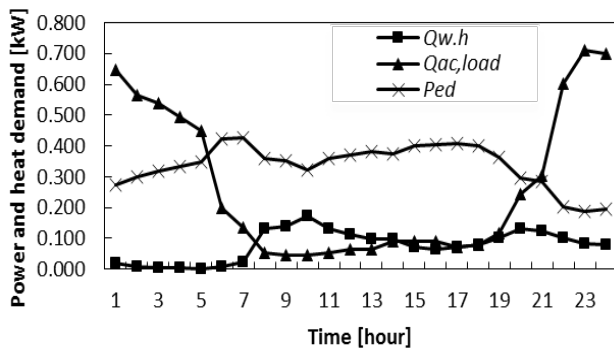


Figure 2. Power, Heating and Cooling Demand in a Single Day

2.4. Data of Micro Gas Turbine in Tri-Generation

MGT-TGS unit that was used in this study consisted of a water-lithium bromide absorption heat pump (AHP), recuperated single shaft MGT and an exhaust heat recovery (EHE) of tube-shell type. Table 1 shows the basic specifications of the AHP, MGT and EHE [2].

Table 1. Basic Specifications of Absorption Heat Pump, MGT, Exhaust Heat Exchanger

| Absorption heat pump (AHP) | | |
|--------------------------------------|----------------|----------------|
| Type | Single effect | |
| Cooling outlet temperature (°C) | 7 | |
| Rated heat medium temperature (°C) | 88 | |
| Standard cooling capacity (kW) | 103 | |
| Heat rejection to cooling tower (kW) | 171 | |
| Heat medium input capacity (kW) | 150 | |
| Micro gas turbine (MGT) | | |
| Rated electrical power output (kW) | 65 | 30 |
| Exhaust mass flow rate (kg/s) | 0.48 | 0.31 |
| Exhaust temperature (°C) | 309 | 273 |
| Fuel input (LHV) (kW) | 224 | 115 |
| Electrical power efficiency | 0.29 | 0.26 |
| Exhaust heat exchanger (EHE) | | |
| Type | Shell and tube | Shell and tube |
| Effectiveness | 0.80 | 0.80 |
| Cold water inlet temperature (°C) | 80 | 80 |
| Cold water mass flow rate (kg/s) | 1.81 | 1.22 |
| Capacity ratio | 0.066 | 0.062 |
| NTU | 1.727 | 1.719 |
| Rated heat recovery (kW) | 105 | 56 |

Two MGT with different power output 30 kW and 65 kW were used in this study. All operation strategies operated at with partial load except for base load. Electrical power Pe , exhaust heat recovery Q_{ehr} , and fuel rate Q_{fuel} for two study MGT can be calculated using the following equations [2]:

$$Pe_{MGT30,PL} = Pe_{MGT30} * LF, \quad (1)$$

$$Pe_{MGT65,PL} = Pe_{MGT65} * LF, \quad (2)$$

$$Q_{ehrMGT30,PL} = Q_{ehrMGT30} * (0.1718 + 0.6529 * LF + 0.1706 * LF^2), \quad (3)$$

$$Q_{ehrMGT65,PL} = Q_{ehrMGT65} * (0.1240 + 0.9707 * LF - 0.1173 * LF^2), \quad (4)$$

$$Q_{fuelMGT30,PL} = Q_{fuelMGT30} * (0.1513 + 0.7824 * LF + 0.06004 * LF^2), \quad (5)$$

$$Q_{fuelMGT65,PL} = Q_{fuelMGT65} * (0.1228 + 0.9766 * LF + 0.1131 * LF^2), \quad (6)$$

where LF is load factor [-].

Exhaust heat recovery is recovered by a shell-tube EHE by using effectiveness-NTU relation. Amount of exhaust heat recovered, Q_{EHR} can be calculated from hot fluid or cold fluid side of the heat recovery system in the following equation [2]:

$$Q_{EHR} = m_{c, cp, c} (t_{c, o} - t_{c, i}) = m_{h, cp, h} (t_{h, i} - t_{h, o}), \quad (7)$$

where c and h represent cold and hot side of the EHE, and $w.h$ and $w.h.loss$ represent water heating and energy loss of heat water respectively.

Study found that the performance of the absorption heat pump is basically affected by input temperature of the heat medium which is hot water supply. Higher temperature of hot water obtained from the exhaust heat recovery, the higher cooling output absorption heat pump and COP. Relation between heat water temperature with cooling output and COP of the absorption heat pump can be expressed as the following equations [5]:

$$Q_{cooling} = 2.2281 * t_{hw} - 125.34, \quad (8)$$

$$COP_{AHP} = 0.0019 * t_{hw} + 0.635, \quad (9)$$

where t_{hm} is heat water temperature or from EHE.

2.5. Environmental Analysis of Trigeneration System

For the environmental analysis two conventional systems; typical and ideal were compared to the MGT-TGS. The typical system worked on a gas turbine whereas the ideal one worked on a Combined Cycle Gas Turbine. The Energy Commission of Malaysia reported the power generation efficiencies for Gas Turbine and Combined Cycle Gas Turbine to be 0.27 and 0.44, respectively. The quantity of global emissions of CO₂, and local emissions of NO_x and CO emissions were considered in this study. Quantities of these gaseous emissions per kWh of generated power by conventional power plants were studied. The equation given below can be used to calculate the gaseous emissions [5]:

$$En = EF_n * Pe, \quad (10)$$

where EF indicates emission factor [kg/kW h] and subscript n indicates any type of pollutant. EF_n for conventional systems are shown in Table 3 [26].

Table 2. Emissions from Distributed Vs. Centralized Generation

| Type of generation | CO ₂ kg/kWh | NO _x kg/kWh | CO kg/kWh |
|----------------------------|---------------------------|---------------------------|--------------|
| Micro gas turbine | 0.725 | 0.0002 | 0.00047 |
| Gas turbine | 0.625 | 0.00029 | 0.00042 |
| Combined cycle gas turbine | 0.363 | 0.000195 | 0.00007 |

It should be noted that 7.5% of transmission and distribution losses was also considered in the net power generation.

For TGS, MGT and boiler are the only source of emissions. Emission increased under partial load and lowest at full load. From the approximation lines, emissions factors of MGT for CO₂, NO_x and CO can be calculated as the following equations [5]:

$$EF_{CO_2,MGT} = 0.00758 * \exp (LF - 20.9 * LF^6), \quad (11)$$

$$EF_{NO_x,MGT} = (23 + 15 * LF - 13 * LF^{LF})^{-3.07}, \quad (12)$$

$$EF_{CO,MGT} = 0.695 + \exp (-2.25 * LF - 5.19 * LF^2), \quad (13)$$

The amount of emissions emitted by the MGT-TGS was calculated and compared to those by the conventional systems by finding out Emissions Reduction Index ERI

which can be expressed as the following equation [28]:

$$ERI_n = \frac{En_{conv} - En_{MGT}}{En_{conv}} \quad (14)$$

3. Results and Discussion

3.1. Capacity of All Equipment

This system operated with trigeneration using MGT-65, EHE and AHP as the core components with different operating strategies. Capacity of each component used in the system is shown in Table 3.

Table 3. Component Required for Each Strategies and Their Capacity

| Power Match | Capacity kWh | Load factor [-] |
|----------------------------|--------------|-----------------|
| Micro gas turbine | 65 | 0.80 |
| Inlet precooling | 6.4 | |
| Exhaust heat recovery | 105 | |
| Absorption heat pump | 103 | |
| Heat storage | 519 | |
| Battery | 1 | |
| Heat Match | | |
| Micro gas turbine | 65 | 0.50 |
| Inlet precooling | 6.4 | |
| Exhaust heat recovery | 105 | |
| Absorption heat pump | 103 | |
| Boiler | 58.1 | |
| External power supply grid | 454 | |
| Mix match | | |
| Micro gas turbine | 65 | 0.92 |
| Inlet precooling | 6.4 | |
| Exhaust heat recovery | 105 | |
| Absorption heat pump | 103 | |
| Heat storage | 519 | |
| Battery | 182 | |
| Base load | | |
| Micro gas turbine | 30 | 1.0 |
| Inlet precooling | 6.4 | |
| Exhaust heat recovery | 56 | |
| Absorption heat pump | 103 | |
| Heat storage | 314 | |
| Boiler | 64 | |
| External power supply grid | 510.7 | |

All operating strategies have the same capacity of MGT, EHE and AHP except base load. Base load and heat match has external power supply grid to get extra electricity cover the insufficient of electricity, boiler is use in base load and heat match to cover insufficient heat supply, battery is used

in power, heat and mix match to store excess power and balance the power demand during day and night, heat storage is used in all match except heat match to balance the heat demand. Average load factor is shown in the Table 3, heat match has lowest average load factor and base load operating strategy has constant 1.0 load factor.

3.2. Power Supply Conditions

Power demand and output conditions of MGT 65 for all the operation strategies are shown in Figure 3. Figure 3a-d represent power match, heat match, mix match and base load operating strategies respectively.

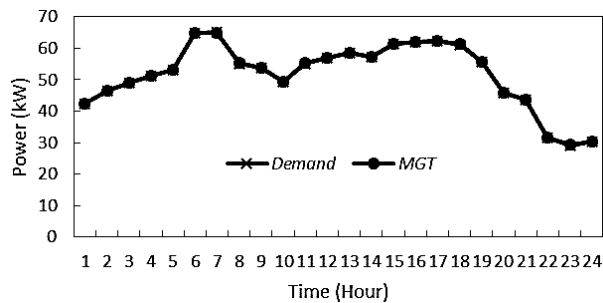


Figure 3a. Power Match

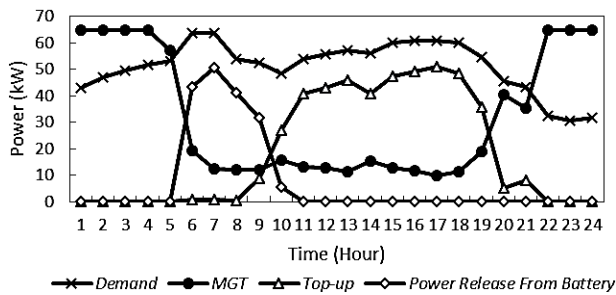


Figure 3b. Heat Match

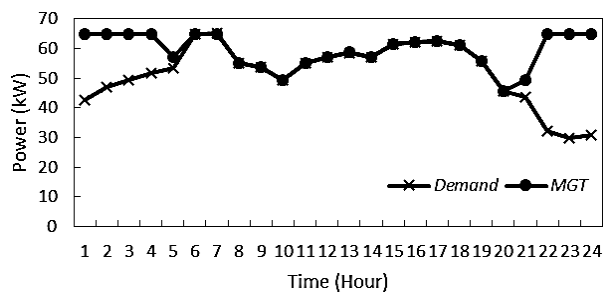


Figure 3c. Mix Match

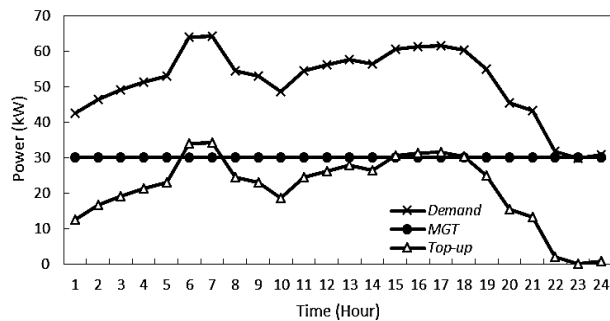


Figure 3d. Base Load

Figure 3a shows the power supply is evenly match with the total power demand of building during day and night with less than 1 kWh excess power produced because load factor was adjusted according to the power demand. Figure 3b shows heat match supply access electricity during night time until early morning 20:00 to 05:00 and from 06:00 to 19:00 only supply at base demand. Battery stored excess power during night time and released in day time. Further insufficient of power supply was supplied by grid. Figure 3c shows mix match power supply evenly match the peak load from 06:00 until 20:00 and supply excess from 20:00 until 05:00, battery was used to store excess power supply and sell it. Figure 3d shows base load operating strategy is supplying power only at base demand, with additional top up from external grid.

3.3. Total Heat Required

Total heat supply by EHE and absorption heat pump are shown in Figure 4. Figure 4a-d represent power match, heat match, mix match, base load operation strategies, respectively.

Figure 4 shows power and mix operating strategies supply enough heating load to meet the requirement, while heat and base load strategies cannot reach the requirement. Heat match supply enough heat from 04:00 to 20:30 because the load factor was adjusted according to heat demand. For all cases, EHE unable to supply enough heat load to absorption heat pump produce cooling load that meet the demand during night. However, extra heat load in the morning can be stored in heat storage and be used to cover the insufficient heating load during night time, as for heat match the insufficient at night time is cover using boiler since this strategy produce no extra heat during day time. Base load using both heat storage and boiler to cover and balance heat demand because the heat produce by 30 kWh MGT is lesser.

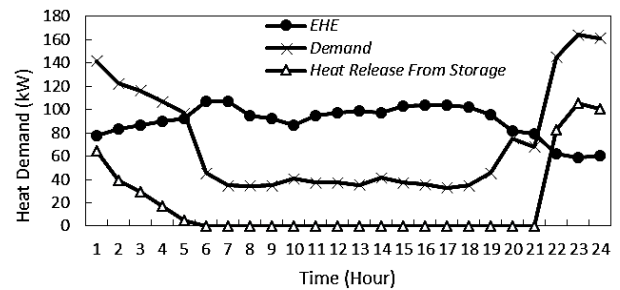


Figure 4a. Power Match

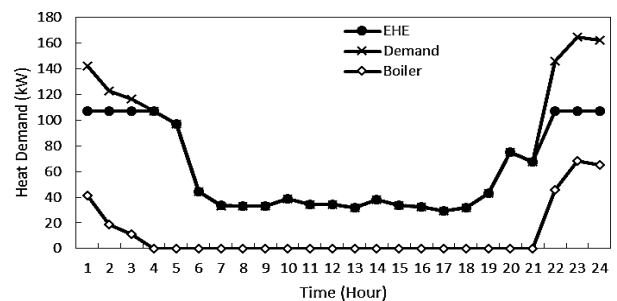


Figure 4b. Heat Match

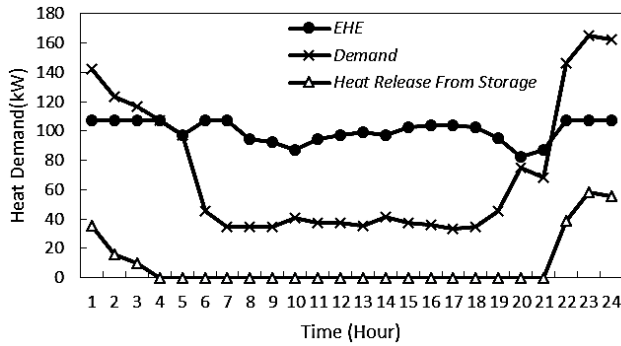


Figure 4c. Mix Match

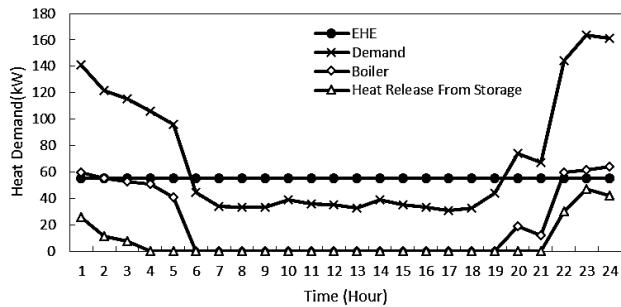
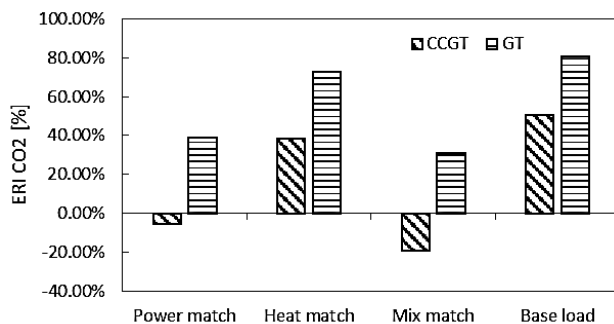


Figure 4d. Base Load

3.4. Emission Performance

Emission results for each operation strategy is shown in Figure 5a-c. Emission reduction index shown are for CO₂, NO_x and CO.

Base load had the highest ERI for CO₂ and CO emissions, heat match had the lowest ERI in CO emission but the highest ERI in NO_x emission. Mix match had the lowest ERI in CO₂ and NO_x. Base load running with low capacity MGT resulted in low CO₂ and CO emissions, whereas heat match that used higher power capacity of MGT but operated at lower load resulting in the lowest NO_x emission. However, it had the highest CO emission. Mix match that used the same power capacity of MGT as the heat match but operated at higher load had the highest CO₂ and NO_x emissions. Base load shows positive ERI for all conditions because it used 30 kW MGT and operated at full load throughout the day.

Figure 5a. Comparison of Emission Reduction Index of CO₂

Results show that compared CCGT, MGT in any operation strategy generally show lower ERI than GT. However, in some operation strategy MGT has lower ERI compared to CCGT. This is because CCGT has higher

power generation efficiency than MGT, 0.44. Thus, the best solution is base load that used smaller MGT. However, economic performance also must be further studied because smaller MGT has higher cost per kW, and it has slightly lower power generation efficiency.

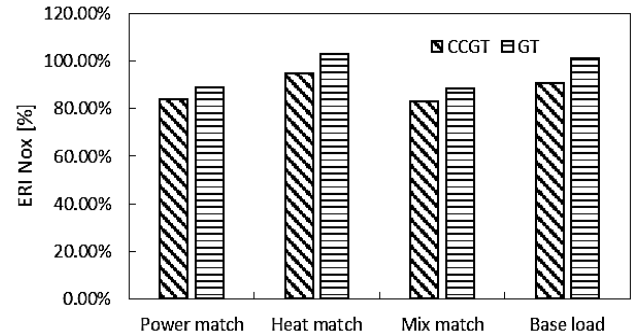
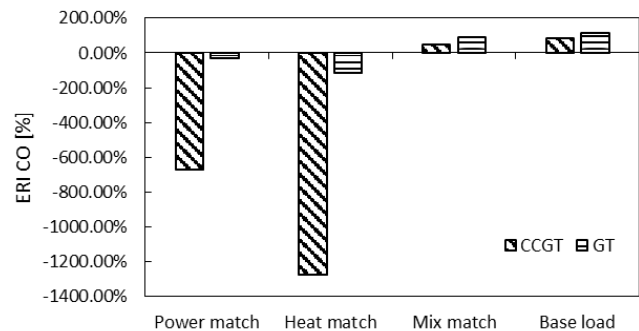
Figure 5b. Comparison of Emission Reduction Index of NO_x

Figure 5c. Comparison of Emission Reduction Index of CO

4. Conclusions

Environmental performance of different operation strategies of MGT-TGS were studied. It was found that all operation strategies able to cover the power and heat demand of residential building through varies addition equipment and configuration of the system. MGT-TGS that compared to a combined cycle gas turbine, cannot reduce CO₂ for power match and mix match, it also cannot reduce CO for power match and heat match. However, MGT-TGS was generally able to reduce all emissions when compared to a gas turbine. Only power and heat match unable to reduce CO, but that is not more than 100%. Among all operation strategies studied, base load emit the lowest emissions and it can reduce all emissions even compared to a combined cycle gas turbine. Thus, the best solution is base load that used smaller MGT. However, economic performance also must be further studied because smaller MGT has higher cost per kW, and it has slightly lower power generation efficiency.

ACKNOWLEDGEMENTS

The authors would like to thank the Ministry of Education

Malaysia for providing research most under FRGS/2/2013/TK06/UMP/02/01.

REFERENCES

- [1] Karen de los Angeles, Tapia-Ahumada, "Understanding the impact of large-scale penetration of micro combined heat & power technologies within energy systems", Doctor of Philosophy thesis, Massachusetts Institute of Technology, United States, 2011.
- [2] Firdaus Basrawi, Takanobu Yamada, Shin'ya Obara, "Theoretical analysis of performance of a micro gas turbine co/trigeneration system for residential buildings in a tropical region", Elsevier Publishing, Journal of Energy and Buildings, vol.67, pp108–117, 2013.
- [3] Online Available: <http://www.iea.org/techinitiatives/end-use-buildings/districtheatingandcooling/>
- [4] Karen de los Angeles, Tapia-Ahumada, "Understanding the impact of large-scale penetration of micro combined heat & power technologies within energy systems", Doctor of Philosophy thesis, Massachusetts Institute of Technology, United States, 2011.
- [5] Firdaus Basrawi, Takanobu Yamada, Shin'ya Obara, "Economic and environmental based operation strategies of a hybrid photovoltaic–microgas turbine trigeneration system", Elsevier Publishing, Journal of Applied Energy, vol. 121, no.1, pp.174-183, 2014.
- [6] Bogdan-andrei Tofan, Ion Erb Noiu, Alin-enver Hoblea, "Comparative analysis of two trigeneration systems for a residential building", M. Eng. Thesis, University of Iași, Romania, 2014.
- [7] Krisztina Uzuneanu, Dan Scarpete. "Energetic and environmental analysis of a micro CCHP system for domestic use", in Proceedings of the 6th IASME/WSEAS international conference on Energy & environment, pp322-327, 2011.
- [8] Donghao Xu, Ming Qu, "Energy, environmental, and economic evaluation of a CCHP system for a data center based on operational data". Elsevier Publishing, Journal of Energy and Buildings, vol.67, pp176-186, 2013.
- [9] Simon Paul Borga, Nicolas James Kelly, "High resolution performance analysis of micro-trigeneration in an energy-efficient residential building", Elsevier Publishing, Journal of Energy and Buildings, vol.67, pp153-165, 2013.
- [10] Hongdong Yu, "The Design, Testing and analysis of a biofuel micro-trigeneration system", Doctor of Philosophy thesis, Sir Joseph Swan Centre for Energy Research Newcastle, United Kingdom, 2012.
- [11] N. Sugiarta, S.A. Tassou, I. Chaer, D. Marriott, "Trigeneration in food retail: An energetic, economic and environmental evaluation for a supermarket application", Elsevier Publishing, Journal of Applied Thermal Engineering, vol. 29, pp2624-2632, 2009.
- [12] Wang Jiang-Jiang, Zhang Chun-Fa, Jing You-Yin, "Multi-criteria analysis of combined cooling, heating and power systems in different climate zones in China", Elsevier Publishing, Journal of Applied Energy, vol.87, pp1247-1259, 2010.
- [13] Masood Ebrahimi, Ali Keshavarz, Arash Jamali, "Energy and exergy analyses of a micro-steam CCHP cycle for a residential building", Elsevier Publishing, Journal of Energy and Buildings, vol.45, pp202–210, 2012.
- [14] K.C. Kavvadias, A.P. Tosios, Z.B. Maroulis, "Design of a combined heating, cooling and power system: Sizing, operation strategy selection and parametric analysis". Elsevier Publishing, Journal of Energy Conversion and Management, vol.51, pp833–845, 2010.
- [15] M.A. Lozano, M. Carvalho, L.M. Serra, "Operational strategy and marginal costs in simple trigeneration systems", Elsevier Publishing, Journal of Energy, vol.34, pp2001–2008, 2009.
- [16] P.J. Mago, L.M. Chamra, J. Ramsay, "Micro-combined cooling, heating and power systems hybrid electric-thermal load following operation", Elsevier Publishing, Journal of Applied Thermal Engineering, vol.30, pp800–806, 2010.