

Exhaust Emissions Characterization of a Single Cylinder Diesel Engine Fueled with Biodiesel-Ethanol-Diesel Blends

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Abstract Due to exponential growth in industrialization, the demand for conventional automotive fuels increases sharply which adversely affects not only the economy but the environment. This necessitates the quest for an alternative fuel. In this research, the blends of ethanol and biodiesel with diesel in varying proportions were carried out to access the emission levels of the blends. The emission levels of carbon dioxide (CO₂), carbon monoxide (CO), hydrocarbon (HC), oxides of nitrogen (NO_x), oxides of sulfur (SO_x) and exhaust gas temperature were investigated under various operating conditions. The tests were carried out on a horizontal single-cylinder, 4-stroke, air-cooled, 3.68 kW engine. The engine was coupled to a manometer and DG eddy current dynamometer with rated power of 3.68 kW at 2600 rpm and was operated at a constant load of 1000 g. The results showed that BED30 fuel gave the best exhaust emission, and a considerable increase in exhaust temperature of the blends compared to diesel was noticed with the exception of BED30 which appeared to be similar to that of diesel at all speed conditions. The results also indicate that the blends gave less CO when compared to diesel. The minimum and maximum reductions of CO of the blends compared to diesel were 9.5% and 49.1% respectively. The emissions of NO_x, SO_x and CO₂ increased with increase in ethanol in the blends. All the findings are in agreement with the results of other researchers.

Keywords Energy, Emission, Engine, Industrialization, Environment, Biodiesel

1. Introduction

The increasing demand for energy in the world increases the consumption of fuels. As a result, access to the conventional fossil fuels is becoming more and more difficult and an increasing trend for the price of oil would be a disaster for the economic recovery. The peaks in oil price are generally related to the countries' concern for energy availability. Apart from economic aspects, the main problem is that fossil fuels are non-renewable. The global interest in "alternative energy" or "renewable energy" sources is due to the mentioned problems in addition to the environmental issues of fossil fuels. Alternative energy is basically a type of energy produced from a source other than conventional fossil fuels, such as, oil, natural gas, and coal.

According to the International Energy Agency's (IEA) world energy outlook, the total energy consumption in the world will increase by a rate of 1.5% per year until 2030. Figure 2 shows the share of different types of energy

resources in this scenario [1]. Fossil fuels such as coal, oil and gas produce huge amounts of CO₂, the major greenhouse gas, when they are used in transportation, electricity or heat production processes. The best alternative energy resources to replace the fossil fuels are renewable energy carriers such as zero-carbon fuels. When renewable energy sources are used to produce energy for transportation, heat and power they produce little or no carbon dioxide. For this reason, they are often referred to as being 'zero carbon'.

Figure 1 demonstrates that in the IEA scenario, the demand for fossil fuels peaks by 2020, and zero-carbon fuels make up a third of the world's primary sources of energy by 2030. Biomass, which already has a large percentage of the total renewable energy sources, is expected to have a strong growth in the coming decades within three key sectors: heat and power generation, transportation biofuels, and other bio-products. The term "bioenergy" is generally used for the energy extracted from biomass resources for the industrial or residential applications, or the extracted liquid fuel, biofuel, to be used in the transportation sector.

Therefore, global consciousness has started to grow to prevent the fuel crisis by developing alternative fuel sources for engine applications and industrial facilities. Many

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research programs are going on to replace diesel fuel with a suitable alternative fuel like biofuels and biodiesel. Nonedible sources, such as neem oil, karanja oil, mahua oil, jatropha oil, simarouba oil etc. are being investigated for biodiesel production. While, sources like sawdust of soft and hard woods are being investigated for bioethanol production.

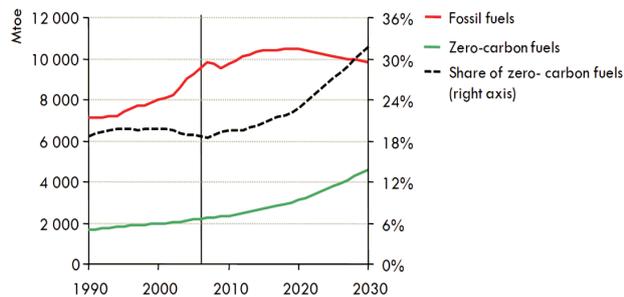


Figure 1. World primary energy demand by fuel type (IEA, 2009)

There is no single solution to global warming, which is primarily a problem of too much heat-trapping carbon dioxide (CO₂), methane and nitrous oxide in the atmosphere. The technologies and approaches outlined by many researchers are all needed to bring down the emissions of these gases by at least 80 percent by 2080 [2]. The uses of biodiesel and biofuel have been found to be of great contributor of the reduction of global warming.

2. Materials and Methods

2.1. Materials/Equipment

2.1.1. Ethanol

The chemical formula for ethanol is C₂H₅OH, sometimes written EtOH or C₂H₆O. It is also known under the names ethyl alcohol or hydroxyethane and is the type of alcohol found in alcoholic beverages. Ethanol is a rather simple organic molecule consisting of a group of carbon and hydrogen atoms, with a hydroxyl group (oxygen and hydrogen atom) attached. The ethanol molecule is small and light, having a molecular weight of 46 g/mol [3].

The ethanol used in the research was produced from sawdust of *Masonia* (*Masonia Altissima*) wood using simultaneous saccharification and fermentation (SSF) processes.

2.1.2. Biodiesel

Biodiesel, by definition, are the methyl esters of long chain fatty acids. When compared to petro diesel, biodiesel has many advantages such that it is a renewable source of fuel, no engine modifications required for using biodiesel blends, lower CO₂ and other exhaust gas emissions, etc. Most commonly used feedstock for biodiesel production currently are canola oil, soybean oil, rapeseed oil and corn oil oil. Since usage of edible oils for biodiesel production raises issues regarding food security especially in the developing

countries, a lot of research are carried out using non-edible oils [4].

The biodiesel was produced from neem oil, which was extracted from neem seeds (*Azadirachta indica*), by transesterification method.

2.1.3. Diesel Fuel

Diesel fuel is a mixture of hydrocarbons obtained by distillation of crude oil. The important properties which are used to characterize diesel fuel include cetane number (or cetane index), fuel volatility, density, viscosity, cold behavior, and sulfur content.

Diesel fuel originated from experiments conducted by German scientist and inventor Rudolf Diesel for his compression-ignition engine he invented in 1892. Diesel originally designed his engine to use coal dust as fuel, and experimented with other fuels including vegetable oils, such as peanut oil, which was used to power the engines which he exhibited at the 1900 Paris Exposition and the 1911 World's Fair in Paris [5].

The diesel fuel used for this research was purchased at AYM Shafa filling station along Abubakar Tafawa Balewa Road, near Yelwa Bauchi, Nigeria.

2.1.4. Automotive Emission Analyzer



Plate 1. Diesel engine test bed



Plate 2. NHA-506EN automotive emission analyzer

Table 1. Test Engine Characteristics

Parameters	Specifications
Engine type	Horizontal single cylinder 4-stroke diesel
Number of cylinders	1
Overall dimension, mm	535 x 326 x 445 (L x W x H)
Bore stroke, mm	75 x 75
Compression ratio	20.5:1 to 22:1
Cooling method	Air cooling by blowing
Method of lubrication	Centrifugal lubrication, combined oil mist and splash
Valve configuration	OHC 2 valves
Oil sump capacity	1.2 liters
Fuel tank capacity	4.5 liters
Maximum output, kW	3.68
Rated speed, rpm	2600
Advance angle injection	20° – 24° before TDC

Source: (TecQuipment TD115 MK-II, 2010)

It is simply used by inserting a probe into the exhaust pipe and press a key to initiate measurements. Readings appear on a back-lit liquid-crystal display on the front of the analyzer.

Emissions of unburned hydrocarbons (HC), concentrations of carbon monoxide (CO) and carbon dioxide (CO₂), oxides of nitrogen (NO_x) and oxides of sulfur (SO_x) were measured using NHA-506EN Automotive Emission Analyzer (Nanhua), which was obtained from the Nigerian Institute of Transport Technology (NITT), Zaria, Kaduna State – Nigeria.

The Engine Test Bed (ETB) and the Automotive Emission Analyzer (AEA) used to conduct the experiments are shown in Plate 1 and Plate 2 respectively, while the details of the ETB are given in Table 1.

2.2. Methods

2.2.1. Blending of the Samples

The biodiesel, the ethanol and the diesel were blended in dry flask equipped with a magnetic stirrer. The biodiesel-ethanol-diesel blends were produced with (% v/v) 5, 10, 15, 20, 25, 30 and 35% ethanol with 85, 80, 75, 70, 65, 60 and 55% diesel, and biodiesel was kept at 10% throughout. Thus, the blends were labeled as BED5, BED10, BED15, BED20, BED25, BED30 and BED35, respectively.

3. Results and Discussions

3.1.1. Exhaust Gas Temperature

The exhaust gas temperatures from the test engine are higher with all the fuel blends than with diesel fuel due to the higher heating value of the blends. The exhaust gas temperature of BED30 appeared to be similar to that of the diesel at all speed conditions, except at 1920 rpm, where it is a bit higher (127°C for the diesel and 142°C for the BED30). The BED30 also appeared to be the best among the blends

due to the lower heating value at all speed conditions. The variation of the exhaust temperature with engine speed tend to be linear, except for BED15, BED20 and BED30, and the highest value was below 200°C at all the speed conditions as shown in Figure 2; this confirmed with the study carried out by [6].

The higher exhaust temperature with the fuel containing higher amount of ethanol is indicative of lower thermal efficiency, less of the energy input in the fuel was converted to useful work, and the remaining heat is given out as exhaust temperature. This agrees with the findings reported by [6] and [7]. However, [8] stated that higher efficiency may be an indicator of better combustion behaviour of the ethanol-diesel blends, resulting in higher temperature and consequently higher exhaust temperature. [8] Explanation for the high exhaust gas temperature is the fact that higher ignition delays results in a delayed combustion and higher exhaust temperature.

3.1.2. Carbon Dioxide (CO₂) Emission at Different Speed Conditions

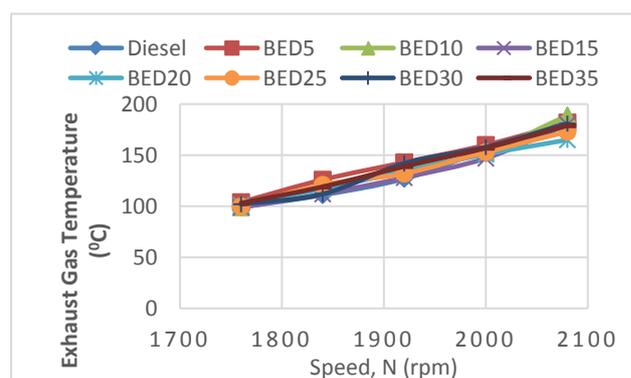


Figure 2. Exhaust gas temperature for the blends and diesel at different speeds

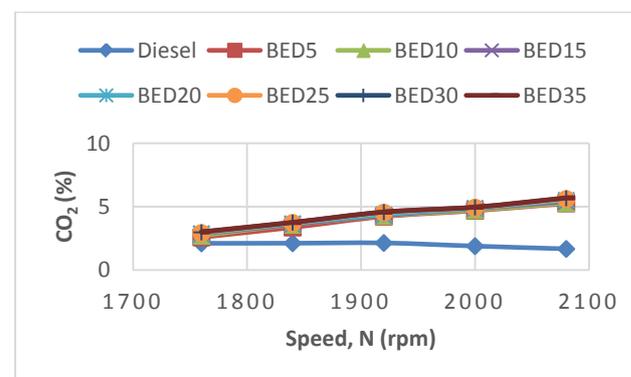


Figure 3. Carbon dioxide (CO₂) emission for the blends and diesel at different speeds

The carbon dioxide emission depends upon the complete combustion of the fuel. Since all the blends have more oxygen content than diesel because of the presence of ethanol, this results in complete combustion. Due to the complete combustion of the fuel blends, carbon dioxide emission also increases. It can be seen from Figure 3 that the

carbon dioxide emission using diesel fuel is lower because of the incomplete combustion as compared to the blends. At all speed conditions, BED30 and BED35 exhibited the highest values of CO₂ emission, because of the complete combustion and are the best among the fuel blends as indicated in Figure 3. The increase in CO₂ is directly proportional to the increase in the amount of ethanol as shown in Figure 2; this is because ethanol contained more oxygen.

Carbon dioxide (CO₂) has great effect on the greenhouse gases, in that it has the tendency to form a blanket effect on the atmosphere which aids the global warming. The CO₂ generally increases with increase in engine speed. At 1760 rpm the CO₂ emission of BED5 is lowest but the diesel has the lowest CO₂ at 2080 rpm. At 2000 rpm the CO₂ emission is almost same for all the blends. [8] Reported more CO₂ emission by the ethanol-diesel blends tested, while [9] found that the CO₂ emissions from a diesel engine were higher. At the highest speed of 2080 rpm the CO₂ emitted for BED30 is 7% as compared to 1.9% at 1760 rpm. CO₂ emission indicates how efficiently the fuel burns inside the combustion chamber.

3.1.3. Carbon Monoxide (CO) Emission at Different Speed Conditions

The results for the carbon monoxide emission of the blends and diesel at different speed conditions are shown in Figure 4. The carbon monoxide emission depends upon the oxygen content and cetane number of the fuel. The blends have more oxygen content than the diesel fuel; so, the blends having more oxygen content are involved in complete combustion process. The maximum carbon monoxide emission was observed at 1760 rpm of the engine speed when run on diesel fuel. So, it can be seen clearly that the fuel blend BED35 gives low carbon monoxide emission than the other fuels at all speed conditions.

The increase of the ethanol percentage in the fuel blend reduces the CO as the BED35 has the least CO. This is possible because of the additional oxygen present in the ethanol, which assists in making the combustion more complete.

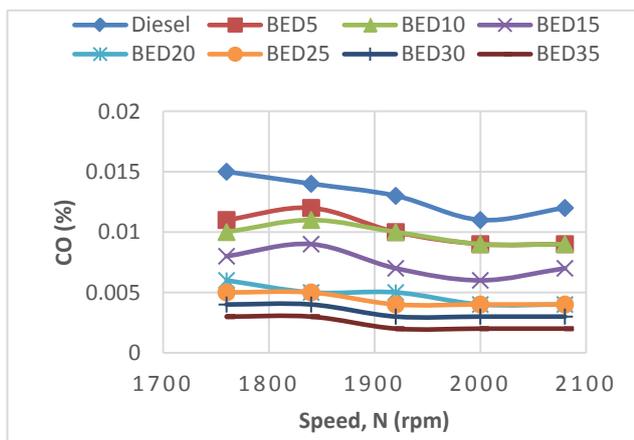


Figure 4. Carbon mono (CO) emission for the blends and diesel at different speeds

Figure 4 illustrates the variation of CO emission with engine speed. It can be seen that at 1760 rpm the BED35 has the least CO emission followed by the BED30. The CO emission is lower at 2000 rpm for all the blends. This shows that with more speed, the engine requires additional oxygen for its combustion which was not readily available with the additional injection of fuel. This tends to increase the emission of CO. This result is in line with the findings of studies conducted by [6], [8], [10] and [11] that show the use of ethanol-diesel and biodiesel fuels result in the reduction of unburned HCs and CO emissions.

The most noticeable reduction appeared at intermediate speed of 1920 rpm. At low speed condition, cylinder temperature was too low, that increased with loading due to more fuel injected inside the cylinder. Findings of [10] indicated that for low speed, CO reduction was practically unaffected by the addition of any fuel to diesel. The study further stated that at elevated temperature, performance of the engine improved with relatively better burning of the fuel resulting in decreased CO.

3.1.4. Hydrocarbon (HC) Emission at Different Speed Conditions

The hydrocarbon emissions of the blends are lower than that of standard diesel due to complete combustion process. When percentage of ethanol increases in the blend, hydrocarbon decreases. Figure 5 clearly shows that fuel blend BED35 gave lower hydrocarbon emissions than the other blends. BED5 at all speed conditions gave an amount of hydrocarbon close to that of BED10, while that of diesel fuel is higher at all the speeds. The HC of diesel decreases with increase in speeds; the trend is the same with the remaining fuel blends. BED30 and BED35 have almost the same values at all speeds, as indicated in Figure 5.

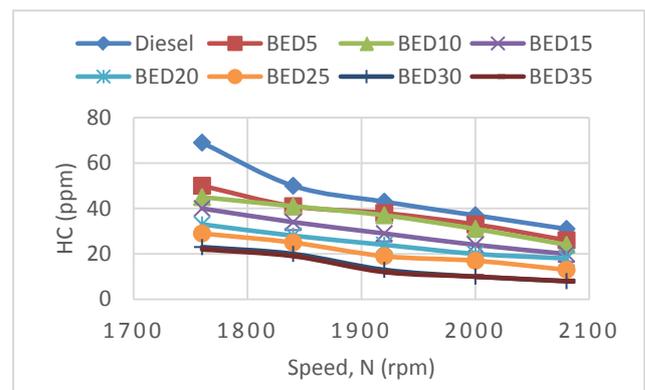


Figure 5. Hydrocarbon (HC) emission for the blends and diesel at different speeds

The HC emissions throughout the study were low. At 1760 rpm, the HC emissions were lower than that at 1840 rpm. However, at 1920 rpm the HC emission dropped further by about 6% for all the blends. On the whole, there was a decrease in the HC with increase in the speed. The result also highlighted the fact that at low speed, the HC emission of the BED35 is considerably less than that of BED30 and both

tend to lower further at 1760 rpm. [8] findings attest reductions in CO and HC emissions for ethanol-diesel blends. The reasons are attributed to the oxygen content of the fuels. Studies by [12] shows that unburned hydrocarbons generally emanate from the regions in a cylinder where there is a poor fuel/air mixture to such a high extent that the ratio exceeds the combustion limit. The BED30 and BED35 showed consistently low level of HC emission when compared to both the rest of the blends and the diesel at all speeds.

3.1.5. Oxides of Nitrogen (NO_x) Emission at Different Speed Conditions

NO_x emissions depend on the oxygen concentration and the combustion time. At all speeds conditions, NO_x emission of the blends is always higher than that of standard diesel due to the oxygen concentration and combustion timing. Since ethanol has very low cetane number, the cetane numbers of the blends are lower than that of standard diesel. This causes increase in the NO_x emission of the blends. The shorter ignition delay could be a reason of increased NO_x emission. BED15 and BED20 show similar values at all speed conditions. Similarly, ED30 and ED35 show the same trend. All the fuel blends have values of NO_x which is above 110ppm at high speed (above 2000 rpm), while that of diesel was below this value at the rate of the same speeds (Figure 6).

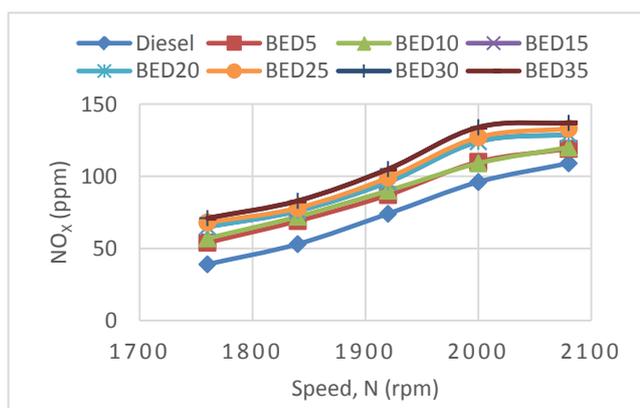


Figure 6. Oxide of Nitrogen (NO_x) emission for the blends and diesel at different speeds

At 2080 rpm, BED35 has more NO_x emission than all the blends, with BED5 having least. As speed is increased, so does all NO_x emission in all the blends. However, the diesel gave the least NO_x emission at speed of 1760 rpm. At low speeds, engine block and cylinder temperature was not very hot therefore little amount of NO_x was produced. But studies by [13] show reduction in NO_x consistently for some tests done. They stated that the emission of NO_x is determined by oxygen concentration, peak pressure, combustion temperature and time. From this, it can be concluded that any change in combustion temperature and stoichiometry will affect the production of NO_x. In summary, it can be seen from Figure 6 that there was a gradual increase of NO_x emission in all the fuels as the speed is increased.

3.1.6. Oxides of sulfur (SO_x) Emission at Different Speed Conditions

Figure 7 shows the results for the oxide of sulfur emission of the blends and diesel at different speed conditions.

Sulfur dioxide (SO₂) is generated from the sulfur present in diesel fuel. The concentration of SO₂ in the exhaust gas depends on the sulfur content of the fuel. Low sulfur fuels of less than 0.05% sulfur are being recommended for most diesel engine applications [14]. It can be seen, from Figure 7, that the SO₂ decreases with increase in the speed of the engine and BED30 and BED35 appeared to be the best candidates with lowest SO₂ emission when compared with diesel and the blends. The decrease of the SO₂ emission as the speed increases indicated that the fuel consumption is low at high speed, and this agrees with the findings reported by [6] and [8]. The SO₂ content of the diesel and the blends is below 13 ppm which indicated a better sulfur content, because most biodiesels contain less than 15 ppm sulfur as reported by [15]. The sulfur limit for highway diesel fuel is 0.015% (15 ppm).

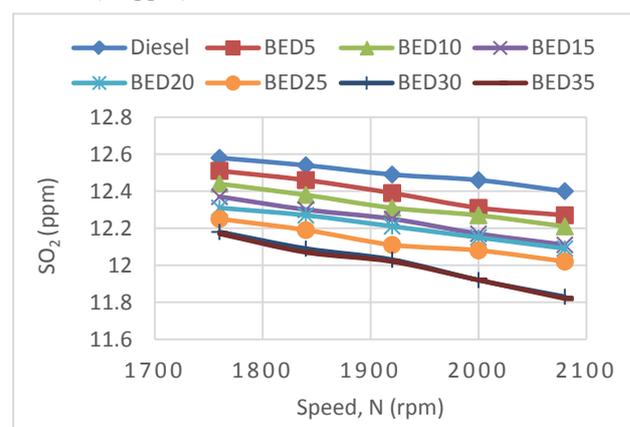


Figure 7. Oxide of sulfur (SO₂) emission for the blends and diesel at different speeds

4. Conclusions

From the experimental investigation the following conclusions may be drawn:

- 1) There was a considerable increase in exhaust temperature with the blends compared to the diesel, but the exhaust temperatures of the blend were higher than that of the diesel fuel in the study. The maximum value of exhaust temperature was obtained with the BED35 (194°C) followed by the BED30 (192°C), at 1760 rpm. However, there is a steady rise of this temperature as the power of the engine is increased;
- 2) The CO₂ generally increases with increase in engine speed. At 1760 rpm the CO₂ emission of BED5 is lowest but the diesel has the lowest CO₂ at 2080 rpm. At 2000 rpm the CO₂ emission is almost same for all the blends, thus, CO₂ emission indicates how efficiently the fuel burns inside the combustion chamber;

- 3) The increase in the amount of ethanol in the fuel blend reduces the CO as the BED35 has the least CO. This is possible because of the additional oxygen present in the ethanol, which assist in making the combustion, will tend to move the combustion process in coming to completion;
- 4) The BED30 and BED35 had shown a consistent low level of HC emission when compared to both the rest of the blends and the diesel at all the speeds;
- 5) At 2080 rpm, BED35 has more NOx emission than all the blends, with BED5 having the least. As the speed is increased, so does all NOx emission in all the blends. However, the diesel gave the least NOx emission at speed of 1760 rpm, because at low speeds, engine block and cylinder temperature was not very hot; therefore, little amount of NOx was produced; and
- 6) The SO₂ decreases with increase in the speed of the engine and BED30 and BED35 appeared to be the best candidates with lowest SO₂ emission when compared with diesel and the blends.

REFERENCES

- [1] IEA report (2009). "World Energy Outlook 2009", *International Energy Agency*, Paris.
- [2] K. R. Senthil and R. Manimaran (2014): "Performance and emission characteristics on 4 stroke single cylinder CI engine using cottonseed bio fuels" *International Journal of Engineering Research & Technology (IJERT)*, 2 (11), pp 36-43.
- [3] J. Martinez-Frias, and H. Francis (2012) "Improving Ethanol Life Cycle Energy Efficiency by Direct Utilization of Wet Ethanol in HCCI Engines," ASME (American Society of Mechanical Engineers).
- [4] M. A. Bote, and H. M Dange,. (2014). "Performance analysis of single cylinder four stroke petrol engine using petrol blended with Thumba oil" ISSN: 2319-8753 Vol. 3, Issue 2 pp34-39.
- [5] D. Ayhan (2008). *Biodiesel: A Realistic Fuel Alternative for Diesel Engines*. Berlin: Springer. pp74. ISBN 1-84628-994-7.
- [6] I. Yahuza, H. Dandakouta, M. E. Ibrahim (2016): Performance Evaluation of Ethanol-Diesel Blend in Compression-Ignition Engine. *Journal of Bioprocess Biotech.* 6: 281. doi: 10.4172/2155 9821.1000281.
- [7] K. Pramanik, (2003) 'Properties and use of jatropha curcas oil and diesel fuel blends in compression ignition engine', *Renewable Energy*, 28, (2), pp239-248.
- [8] A. C. Hansen, R. H. Hornbaker, Q. Zhang, P. W. L. Lyne, (2011). On farm Evaluation of Diesel Fuel Oxygenated with Ethanol. ASAE Paper No. 03-5620. ASAE.
- [9] A. K. Agarwal, (2007). Biofuels (alcohols and biodiesel) applications as fuels for internal combustion engines. *Progress in Energy and Combustion Science*, 33, pp233-271.
- [10] S. Kalligeros, F. Zannikos, S. Stournas, E. Lois, G. Anastopoulos, C. Teas, and F. Sakellaropoulos, (2003) 'An investigation of using biodiesel/marine diesel blends on the performance of a stationary diesel engine', *Biomass and Bioenergy*, 24, (2), pp. 141-149.
- [11] W. Marshall, L. G. Schumacher and S. Howell, (1995) *Engine Exhaust Emissions Evaluation of a Cummins L10e When Fuelled with a Biodiesel Blend*. NIPER-BDM Oklahoma, University of Missouri MARC IV.
- [12] A. Monyem and H. J. Van Gerpen, (2001) 'The effect of biodiesel oxidation on engine performance and emissions', *Biomass and Bioenergy*, 20, (4), pp317-325.
- [13] S. Puhan, N. Vedaraman, B. V. B. Ram, G. Sankarnarayanan and K. Jeychandran (2005a) 'Mahua oil (Madhuca Indica seed oil) methyl ester as biodiesel-preparation and emission characteristics', *Biomass and Bioenergy*, 28, (1), pp87-93.
- [14] IEA (International Energy Agency), "Biofuels for Transport — An International Perspective," (2014)., <http://www.iea.org/textbase/nppdf/free/2014/biofuels2014.pdf>.
- [15] M. Mittelbach and C. Remschmidt (2006) *Biodiesel: The comprehensive handbook*. Austria: Martin Mittelbach.